

AMENDMENTS TO THE SPECIFICATION

[0001] A rapid coupling ~~having the features specified in the preamble of claim 1~~ is disclosed in EP 0 467 381 A1. ~~There, an~~ An engagement section is provided on the pipe nipple ~~on~~ in the form of an annular bead or groove which cooperates with a resilient locking element disposed inside the ~~bush~~ sleeve in such a way that the pipe nipple is retained in the ~~bush~~ sleeve by engagement between the locking element and the engagement section.

[0002] When the pipe nipple is pushed into the ~~bush~~ sleeve, latching takes place between the locking element and the engagement which is as such audible and tangible. However, if mounting is done automatically or under ~~unfavourable~~ unfavorable working conditions, audible or tangible latching as an indicator of a complete locking has to be ruled out. Also in a subsequent inspection, the locking state can only be checked by trying to pull the pipe nipple out of the ~~bush~~ sleeve, which is ~~labour~~ labor-intensive and is not readily possible in the case of fully assembled machines.

Summary of the Invention

[0003] A general object of the invention is to overcome, at least in part, disadvantages as occur in comparable rapid couplings of the prior art. A more specific object of the invention may be seen in providing a rapid coupling in which the locking state between the pipe nipple and the ~~bush~~ sleeve can easily be ascertained.

[0004] This object is met by the present rapid coupling ~~defined in claim 1~~. ~~In the arrangement of claim 1, the~~ The engagement section (groove or projection) is outside the ~~bush~~ sleeve and is thus visible when the pipe nipple and the ~~bush~~ sleeve are not properly coupled and thus not locked. Thus, the engagement section, in addition to its actual latching function with the locking element, assumes the additional task as an indicator of the locking state of the rapid coupling.

[0005] ~~The development of claim 2 is particularly advantageous in that the~~ compression spring again serves a double purpose, namely to bias the engagement between the locking element of the ~~bush~~ sleeve and the engagement section of the pipe nipple toward a defined position, on the one hand, and to ensure that the pipe nipple is pushed out of the ~~bush~~ sleeve in the unlocked state, so that the engagement section reliably fulfils the said indicator function, on the other hand.

[0006] ~~Claims 3 to 5, and claims 6 and 7 relate to expedient embodiments of the~~The
present locking mechanisms, which are uncomplicated in production, ensure reliable
locking and yet allow simple uncoupling.

Brief Description of the Drawing

[0007] Embodiments of the invention are explained in more detail below with
reference to the drawing, in which:

[0008] Fig. 1 shows a rapid coupling, partly in axial section, in the coupled and
locked state;

[0009] Fig. 2 is an enlarged detailed view of the locking region of the rapid coupling
according to Fig. 1;

[00010] Fig. 3 shows the same coupling in the pushed-together, but not yet locked
state; and

[00011] Fig. 4 is an illustration similar to Fig. 1 of a rapid coupling with a locking
arrangement of different design.

Detailed Description of Embodiments

[00012] The rapid coupling shown in Fig. 1 comprises a ~~bush-sleeve~~ 10 having a
through-hole 11 and a pipe nipple 12 which can be inserted into said ~~bush-sleeve~~ 10
and is to be coupled to the ~~bushsleeve~~. The bush-sleeve 10 is provided, at its end
remote from the insertion end 13 for the pipe nipple 12, with an external thread 14
for screwing into a machine housing, for example a motor-vehicle engine block and,
on a further part of its outer surface, with a hexagon 15. Instead of a screw
connection, the ~~bush-sleeve~~ 10 may be formed as an insert part to be inserted into a
corresponding hole of the machine housing and fastened therein by material
displacement, as described in EP 0 467 381 A1.

[00013] The pipe nipple 12 is of cylindrical shape over its length to be inserted into the
~~bush-sleeve~~ 10 and, in the embodiment of ~~to~~ Figs. 1 to 3, has an annular recess or
groove 17 at some distance from its insertion end 16. In the embodiment shown,
the pipe nipple 12 is curved outside the ~~bush-sleeve~~ 10.

[00014] A helical compression spring 19 is disposed between the insertion end 16 of
the pipe nipple 12 and an annular stop 18 formed in the interior of the through-hole
11 of the ~~bush-sleeve~~ 10, the outer diameter of the compression spring 19, in the
compressed state, being approximately equal to the inner diameter of the through-

hole 11. In the embodiment shown, the stop 18 is located at the end of the ~~bush~~ sleeve 10 remote from the insertion end 13.

[00015] Furthermore, the ~~bush-sleeve~~ 10, in the region where the cylindrical section 20 between the insertion end 16 and the groove 17 of the pipe nipple 12 is located in the coupled state, has an annular recess 21 into which a sealing O-ring 22 is inserted.

[00016] Serving to lock the pipe nipple 12 in the ~~bush-sleeve~~ 10 in the coupling state shown in Figs. 1 and 2, is a locking ring 23 which is resilient in the radial direction, is split in the circumferential direction, and is dimensioned in such a way that its inner diameter in the released state is smaller than the outer diameter of the pipe nipple 12. The locking ring 23 is located in an annular recess 24 which is formed in the ~~bush-sleeve~~ 10 close to its insertion end 13 and has three successive regions 25...27 of different diameters.

[00017] The inner region 25 has an inner diameter which is slightly larger than the diameter of the pipe nipple 12 plus twice the radial thickness of the locking ring 23. The inner diameter of the ~~centre-center~~ region 26 of the annular recess 24 corresponds to the outer diameter of the groove 17 provided in the pipe nipple 12 plus twice the radial thickness of the locking ring 23. The inner diameter of the outer region 27 of the annular recess 24 is smaller than that of the ~~centre-center~~ region 26.

[00018] If the pipe nipple 12 ~~is pushed sufficiently far~~ with its end section 20 is pushed sufficiently far into the through-hole 11 of the ~~bush-sleeve~~ 10, the groove 17 formed in the pipe nipple 12 passes into the region of the annular recess 24 provided in the ~~bush-sleeve~~ 10, and the locking ring 23 enters the groove 17 in which it latches in place due to its bias.

[00019] When the pipe nipple 12 is released, the compression spring 19, which is compressed when said pipe nipple 12 is being pushed in, seeks to push the pipe nipple 12 outward, the locking ring 23 being urged into the ~~centre-center~~ region 26 of the annular recess 24 until it abuts against the step between the ~~centre-center~~ region 26 and the outer region 27 of the annular recess 24 in the position shown in Fig. 2. This is the fully coupled and locked position, in which the groove 17 of the pipe nipple is located completely within the ~~bush-sleeve~~ 10.

[00020] An unlocking tool consisting of two tubular halves, which may have the form shown in Fig. 12 of EP 0 467 381 A1 and is not shown here, may be used for releasing the coupling. The two tubular halves of this tool are so dimensioned that they can be passed through the gap between the outer region 27 of the annular recess 24 and the outside of the pipe nipple 12. If the pipe nipple 12 is then pushed inward against the force of the compression spring 19 until the locking ring 23 faces the wider, inner region 25 of the annular recess 24, the locking ring 23 can be lifted out of the groove 17 of the pipe nipple by means of the tool, whereupon the pipe nipple 12 can be pulled out of the through-hole 11 of the ~~bush-sleeve~~ 10.

[00021] As shown in Fig. 1 of EP 0 467 381 A1, the unlocking tool may have the form of a sleeve movable on the pipe nipple and having a wall thickness corresponding to the gap between the outer region 27 of the annular recess 24 and the outside of the pipe nipple 12. In order not to damage the coupling, such a sleeve is preferably made of plastic.

[00022] In the state shown in Fig. 3, the end section 20 of the pipe nipple 12 is still engages in the bush-sleeve 10, and the O-ring 22 may bear against the end section 20 and effect a seal. In this state, however, the coupling is not locked, so that an unintentional relative movement between the ~~bush-sleeve~~ 10 and the pipe nipple 12 may lead to leakage or even to complete release of the coupling. This non-locked condition can be readily ~~recognised~~ recognized visually, with the naked eye or by means of an imaging device, because the groove 17 of the pipe nipple 12 is visible outside the ~~bush-sleeve~~ 10.

[00023] In the second embodiment shown in Fig. 4, the pipe nipple 12, instead of having the groove, has a projection 28 which can be formed, for example, as an annular bead by axial upsetting of the pipe nipple. In this case, the annular recess 34 provided in the ~~bush-sleeve~~ 10 has two regions 35, 37, the inner diameter of the inner region 35 being slightly larger than the outer diameter of the annular bead 28 plus twice the radial thickness of the locking ring 23. The inner diameter of the outer region 37 of the annular recess 34 corresponds to the outer diameter of the bead 28; to be precise, it is only slightly larger than the latter, so that the pipe nipple 12 can easily be inserted. The outer region 37 and the outer surface of the pipe nipple 12 form a gap for inserting the above mentioned unlocking tool.

[00024] When the end section **20** of the pipe nipple **12** is pushed in against the force of the compression spring **19**, the locking ring **23**, held in place by the inner end face of the annular-recess region **35**, is lifted over the annular bead **28**. When the pipe nipple **12** is released, the compression spring **19** causes the locking ring **23** to be pressed against the opposite end face of the annular-recess region **35** by the bead **28** and the locking is effected.

[00025] If the pipe nipple **12** is not inserted sufficiently far into the ~~bush-sleeve~~ so that no locking is not achieved, the compression spring **19** pushes the pipe nipple **12** out of the bush **10** to such an extent that the annular bead **28** remains visible outside the bush.

[00026] Since the annular locking recess **24** or **34** in both embodiments is located in direct proximity to the insertion end **13** of the bush **10**, the groove **17** or bead **28** is visible outside the ~~bush-sleeve~~ **10** if no latching is effected. The groove **17** or bead **28** may be located, however, close to the insertion end **13** of the ~~bush-sleeve~~ **10**.

[00027] The compression spring **19**, which defines the latched locking position shown in Figs. 1, 2 and 4, is so dimensioned that, in the unlocked state, it pushes the pipe nipple **12** out of the ~~bush-sleeve~~ **10** to such an extent that the groove **17** or bead **28** is located at some distance from the insertion end **13** of the ~~bush-sleeve~~ **10** and is therefore in any case clearly visible.

List of Reference Numbers

10	Bush
11	Through-hole
12	Pipe nipple
13	Insertion end of the bush 10
14	External thread
15	Hexagon
16	Insertion end of the pipe nipple 12
17	Groove in the pipe nipple 12
18	Step
19	Compression spring
20	End section
21	Annular recess for O-ring 22
22	O-ring
23	Locking ring
24	Annular recess in the bush 10
25	Inner region of the annular recess 24
26	Centre region of the annular recess 24
27	Outer region of the annular recess 24
28	Annular bead
34	Annular recess in the bush 10
35	Inner region of the annular recess 34
37	Outer region of the annular recess 34